

(19)  Eur pâisches Patentamt  
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Office eur péen des brevets



(11) EP 0 661 150 B1

(12) **EUROPEAN PATENT SPECIFICATION**

(45) Date of publication and mention  
of the grant of the patent:  
09.06.1999 Bulletin 1999/23

(51) Int. Cl.<sup>6</sup>: **B29D 30/24**(21) Application number: **94120825.8**(22) Date of filing: **28.12.1994****(54) Inner supporting unit for toroidal carcasses**

Innere Trägereinheit für torusförmige Reifenkarkassen

Dispositif de support intérieur pour carcasses toroidales de pneus

(84) Designated Contracting States:  
DE ES FR GB IT

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(30) Priority: **31.12.1993 IT TO931027**

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(43) Date of publication of application:  
05.07.1995 Bulletin 1995/27

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**Description**

[0001] The present invention relates to an inner supporting unit for toroidal carcasses.

[0002] European Patent Applications publication nos. 0549868 and 0549869 and European Patent Applications nos. 93110346.9 and 93111228.8 relate to the formation of a toroidal carcass comprising two beads with respective fillers; two sidewalls, each defined by a succession of loops formed using at least one cord and enclosing a respective bead and filler; and an intermediate annular portion preferably reinforced externally by a tread belt normally comprising reinforcing wires.

[0003] The above European Applications also relate to the formation of a toroidal carcass of the above type using a reinforcing structure defined by the two beads, which are subsequently incorporated into the carcass; and by an annular element located centrally and radially outwards in relation to the beads for supporting an intermediate annular portion of the carcass and for defining, together with the beads, two annular openings arranged on opposite sides of the annular element an through which part of the carcass extends. Once the carcass is complete, said annular element is required to support it during the formation of a reinforced tread belt on the outer surface of the carcass, and as the reinforced carcass is inserted inside an annular tread formed on the inner surface of a toroidal forming mold as described in European Patent Application publication n. 0540048. Finally, the annular element must be removed from the carcass for finishing it internally by inserting at least an impermeable innerliner.

[0004] US-A-2325001 disclosed a collapsible drum for the manufacture of pneumatic tire casings. The above collapsible drum is of the segmental type and comprises two groups of radially movable sections of which one group is movable axially, as a unit, in relation to the other group in order to allow the drum to collapse to the smallest possible compass. The above collapsible drum may be associated with two annular bead supports arranged on opposite sides of the drum.

[0005] The above collapsible drum is designed to manufacture tires of relatively small bead diameter as compared to cross sectional diameter, for example, the tires of airplane landing wheels and, owing to the absence, between the drum in its expanded configuration and the annular bead supports, of the aforementioned annular openings, is not suitable for the manufacture of a toroidal carcass of the aforementioned type, wherein each sidewall is defined by a succession of loops formed using at least one cord and enclosing a respective bead and filler.

[0006] It is an object of the present invention to provide a highly straightforward, functional supporting unit designed to support the beads during formation of the carcass, formation of the tread belt, and insertion of the carcass inside the tread in said toroidal forming mold; to define said inner annular element; and to enable

removal of the annular element through one of the two beads upon completion of the above operations.

[0007] According to the present invention, there is provided an inner supporting unit for the production of a toroidal carcass comprising an intermediate annular portion, two beads on either side of the intermediate annular portion and two sidewalls, each defined by a succession of loops formed using at least one cord and enclosing a respective bead; the unit comprising a number of ring segments movable into an operating position wherein each segment contacts and is aligned with two adjacent segments, to define a first annular supporting element for the intermediate annular portion of the toroidal carcass; and two rings of supporting members on either side of said first annular element and coaxial with each other and with the axis of the first annular element, movable into an operating position for supporting said two beads; said first annular supporting element being located radially outwards of, and centered axially in relation to, said two rings of supporting members in said operating position, so as to define, with the same, two annular openings; characterized by the fact that the first annular element comprises at least a first and second number of said segments; a first and second transmission being connected respectively to said first and second number of segments, for moving the respective number of segments between said operating position and an idle position wherein said two numbers of segments respectively define a second and third substantially annular element aligned with each other along said axis, and having an outside diameter smaller than the inside diameter of said beads; two actuating devices being connected to a respective one of said two rings of supporting members, for moving the respective ring of supporting members between said operating position and an idle position, wherein said two rings of supporting members have an outside diameter smaller than the inside diameter of said beads.

[0008] The invention will now be described by way of example with reference to the accompanying drawings, in which:

Figures 1 and 2 show partial views in perspective, with parts removed for clarity, of a preferred embodiment of the supporting unit according to the present invention in two different operating positions;

Figure 3 shows a horizontal axial section of the Figure 1 and 2 unit in the Figure 1 position;

Figure 4 shows a horizontal axial section of the Figure 1 and 2 unit in the Figure 2 position;

Figure 5 shows a partially sectioned front view of the Figure 4 unit with parts removed for clarity.

[0009] Number 1 in Figure 4 indicates an inner supporting unit for a carcass 2 substantially formed as described in European Patent Application publication n. 0549868 and fittable inside a forming ring 3 housing a

tread (not shown) and having detachable lateral walls 3a substantially as described in European Patent Application publication n. 0540048.

[0010] Carcass 2 comprises two beads 4 with respective fillers; two sidewalls 5, each defined by a succession of loops formed using at least one cord and enclosing a respective bead 4 and respective filler; and an intermediate annular portion 6 which may be reinforced externally by a tread belt (not shown).

[0011] Unit 1 comprises a central tubular guide shaft 7 projecting from a support (not shown) and connected to a motor 8 so as to rotate about its axis 9. Shaft 7 is defined externally by a cylindrical surface 10, and partially houses a drive 11 comprising a first tubular screw 12 coaxial with axis 9, extending along an initial portion of shaft 7, and connected to a motor 13 housed inside said support (not shown), so as to rotate about axis 9; and a second screw 14 extending along screw 12, coaxially with axis 9, and connected for rotation to screw 12. Screw 14 is connected to a motor 15 housed inside said support (not shown), so as to rotate about axis 9, and presents an end portion projecting from the free end of screw 12 towards the free end of shaft 7, and fitted in rotary manner to shaft 7 via the interposition of bearings 16.

[0012] Shaft 7 is fitted in sliding manner with two tubular slides 17 and 18 coaxial with axis 9 and presenting respective inner appendixes 19 parallel to axis 9 and engaging in transversely sliding manner respective axial openings 20 formed through shaft 7. Appendixes 19 and respective slides 17 and 18 constitute the output members of drive 11, which also comprises two nut screws 21 and 22 fitted respectively to screws 12 and 14 and connected integral with the inner ends of respective appendixes 19.

[0013] Unit 1 also comprises a number of ring segments 23 connected to drive 11 and movable into an operating position (Figures 2, 4 and 5) wherein each segment 23 contacts and is aligned with two adjacent segments 23 to define an annular element 24 coaxial with axis 9 and supporting intermediate annular portion 6 of toroidal carcass 2. Unit 1 also comprises two rings 25 and 26 of supporting elements 27, located on either side of annular element 24, coaxial with each other and with axis 9, for supporting the two beads 4 of carcass 2.

[0014] More specifically, and as shown in Figures 3 and 4, each supporting element 27 is substantially U-shaped and fitted to a respective actuating device indicated 28 for ring 25, and 29 for ring 26. Actuating devices 28 are equally spaced about the inner flange 30 of a ring 31 fitted in axially sliding manner to the open end of a cylindrical lateral wall 32 of a cup-shaped body 33 surrounding the portion of shaft 7 extending between said shaft support (not shown) and annular element 24. Body 33 is coaxial with axis 9, and comprises an end wall 34 perpendicular to axis 9 and fitted through with a sleeve 35 coaxial with axis 9 and force fitted on to the end portion 36 of shaft 7 opposite the free end of the

shaft. Each actuating device 28 comprises a bracket 37 integral with flange 30 and parallel to axis 9; and a rocker arm 38 pivoting on the free end of respective bracket 37 about a pin 39 perpendicular to axis 9. Rocker arm 38 presents a first arm fitted on its free end with a respective supporting element 27; and a second arm connected by its free end to the output of a linear actuator 40 interposed between rocker arm 38 and ring 31, and selectively operable for moving respective supporting element 27 between an inner idle position (Figure 3) and an outer operating position (Figure 4). The axial position of ring 31 in relation to body 33 is defined by a linear actuator 41 interposed between wall 32 and flange 30, and operating parallel to axis 9 for moving respective bead 4 between a carcass forming position (Figure 4) and a carcass shaping position (not shown).

[0015] Each actuating device 29 comprises a cylinder 42 extending radially outwards from a ring 43 fitted in axially movable manner to shaft 7, close to the free end of the shaft. Each cylinder 42 comprises a radial output rod 44 fitted with a respective supporting element 27, and is selectively operable for moving respective supporting element 27 between an inner idle position (Figure 3) and an outer operating position (Figure 4). The axial position of ring 43 in relation to shaft 7 is defined by a linear actuator 45 which is interposed between ring 43 and a bracket 46 on a cap 47 fitted integral with the free end of shaft 7, and operates parallel to axis 9 for moving respective bead 4 between a carcass forming position (Figure 4) and a carcass shaping position (not shown).

[0016] As shown clearly in Figures 3 and 5, alternate segments 23 of annular element 24 define two numbers 48 and 49 of segments 23 connected to drive 11 via the interposition of respective transmissions 50 and 51 respectively connected to slides 17 and 18, and which provide for moving respective numbers 48 and 49 of segments 23 between said operating position and an idle position (shown in Figure 3 and by the dotted line in Figure 5) wherein segments 23 of said two numbers 48 and 49 define respective substantially annular elements 52 and 53 aligned along axis 9 and the outside diameters of which are smaller than the inside diameter of beads 4.

[0017] As shown more clearly in Figures 3 and 4, for each respective segment 23, each transmission 50 and 51 comprises an articulated parallelogram 54 movable in relation to shaft 7 in a radial plane in relation to axis 9, and in turn comprising a connecting rod 55 consisting of an inner appendix of respective segment 23; a frame 56 (indicated 56a for segments 23 in number 48, and 56b for segments 23 in number 49) consisting of a blade lying in a plane through axis 9 and integral with the outer surface of respective slide 17, 18; and two cranks 57 and 58 connecting the opposite ends of connecting rod 55 to the corresponding ends of frame 56.

[0018] For each respective segment 23, each transmission 50 and 51 also comprises an activating device

59 (indicated 59a for segments 23 in number 48, and 59b for segments 23 in number 49) connected to crank 57, for moving connecting rod 55 and respective segment 23 between said idle and operating positions. Each device 59 comprises a rocker arm 60 (indicated 60a for segments 23 in number 48, and 60b for segments 23 in number 49) pivoting on respective frame 56 about a pin 61 perpendicular to axis 9, and presenting two arms on either side of pin 61, the first of which defines a sector gear 62 meshing with a gear 63 rotating with crank 57 in relation to frame 56, and the second of which supports in idle manner a tappet roller 64 mating with a cam 65 (indicated 65a for segments 23 in number 48, and 65b for segments 23 in number 49) formed in a plate 66 (indicated 66a for segments 23 in number 48, and 66b for segments 23 in number 49) fixed in relation to shaft 7.

[0019] More specifically, plate 66a is fitted integral with the inner surface of wall 32; respective cam 65a presents a first portion 67 parallel to axis 9, and a second portion 68 in the form of a ramp converging towards axis 9; and frame 56a extends from respective slide 17 towards the free end of shaft 7, and supports a transmission 69 comprising a chain 70 looped about respective gear 63, and about a further gear 71 meshing with a gear 72 fitted idly to frame 56a and meshing with sector gear 62 of respective rocker arm 60a. Plate 66b, on the other hand, is fitted directly on to shaft 7, is much shorter than plate 66a, and presents a cam 65b comprising only a ramp similar to portion 68 of cam 65a. Also, sector gear 62 of rocker arm 60b meshes directly with respective gear 63.

[0020] Operation of unit 1 will now be described as of an initial position (Figure 3) wherein the two numbers 48 and 49 of segments 23 define respective annular elements 52 and 53 aligned along axis 9, with element 52 withdrawn along shaft 7 in relation to element 53, and substantially inside flange 30 of ring 31. For this purpose, slides 17 and 18 are withdrawn by drive 11 towards portion 36 of shaft 7.

[0021] In the above initial position, supporting elements 27 in rings 25 and 26 are set to the inner idle position by respective actuating devices 28 and 29, and at the same time are set by actuators 41 and 45 to the maximum distance along axis 9, i.e. to the forming position.

[0022] In connection with the above, it should be pointed out that, when slide 17 is moved towards the free end of shaft 7, roller 64 of activating device 59a initially travels along straight portion 67 of cam 65a, so that the whole of annular element 52 travels forward and remains in the initial position until roller 64 reaches ramp portion 68. At this point, as slide 17 continues moving forward, roller 64 travels along portion 68 towards axis 9, so that rocker arm 60a and respective sector gear 62 rotate anticlockwise (in Figures 3 and 4) about respective pin 61; gear 72 is rotated clockwise; and chain 70, respective gear 63, and respective cranks

57 and 58 of respective parallelogram 54 rotate anticlockwise, so that respective connecting rod 55 and respective segment 23 move away from axis 9 into the outer operating position in Figure 4.

5 [0023] Similarly, when slide 18 is moved towards the free end of shaft 7, roller 64 of activating device 59b travels along the ramp portion of cam 65b away from axis 9, so that rocker arm 60b and respective sector gear 62 rotate anticlockwise (in Figures 3 and 4) about

10 respective pin 61; and respective gear 63 and respective cranks 57 and 58 of respective parallelogram 54 rotate clockwise, so that respective connecting rod 55 and respective segment 23 move away from axis 9 into the outer operating position in Figure 4.

15 [0024] In other words, when motors 13 and 15 are operated as of said initial position in Figure 3, slides 17 and 18 are moved differently but both towards the free end of shaft 7, so that segments 23 are all moved into the outer operating position; and the travel distance of slides 17 and 18 is so calculated that segments 23 in

20 said two numbers 48 and 49 are aligned in a plane perpendicular to axis 9, to form annular element 24.

[0025] Formation of annular element 24 is followed immediately by extraction of supporting elements 27 - which engage respective beads 4 prior to reaching the outer operating position - and by weaving or interconnection of carcass 2.

[0026] In connection with the above, it should be pointed out that annular element 24 formed by segments 23 in the operating position presents a relatively small amount of clearance between adjacent segments 23, and, as shown clearly in Figure 4, rollers 64 and hence slides 17 and 18 are stopped short of the end-of-travel position. When inserting carcass 2 inside forming ring 3, in much the same way as for inserting the inner ring of a bearing inside the outer ring, rollers 64 are first backed up further to slightly reduce the outside diameter of carcass 2; once the carcass is inserted inside forming ring 3, actuators 41 and 45 are activated to bring beads 4 slightly closer together into the shaping position; and rollers 64 are moved forward into the end-of-travel position to shape carcass 2 against the tread (not shown) inside ring 3, and take up any slack in sidewalls 5 caused when beads 4 are brought together.

30 35 40 45 [0027] Carcass 2 is removed from unit 1 by restoring slides 17 and 18, devices 28 and 29, and actuators 41 and 45 to their original positions, and by axially moving ring 3 so as to withdraw unit 1 completely from carcass 2 through the bead 4 closest to cup-shaped body 33.

#### 50 Claims

55 1. An inner supporting unit for the production of a toroidal carcass (2) comprising an intermediate annular portion (6), two beads (4) on either side of the intermediate annular portion (6) and two sidewalls (5), each defined by a succession of loops formed using at least one cord and enclosing a

respective bead (4); the unit (1) comprising a number of ring segments (23) movable into an operating position wherein each segment (23) contacts and is aligned with two adjacent segments (23), to define a first annular supporting element (24) for the intermediate annular portion (6) of the toroidal carcass (2); and two rings (25, 26) of supporting members (27) on either side of said first annular element (24) and coaxial with each other and with the axis (9) of the first annular element (24), movable into an operating position for supporting said two beads (4); said first annular supporting element (24) being located radially outwards of, and centered axially in relation to, said two rings (25, 26) of supporting members (27) in said operating position, so as to define, with the same, two annular openings; characterized by the fact that the first annular element (24) comprises at least a first (48) and second (49) number of said segments (23); a first (50) and second (51) transmission being connected respectively to said first (48) and second (49) number of segments (23), for moving the respective number (48)(49) of segments (23) between said operating position and an idle position wherein said two numbers (48, 49) of segments (23) respectively define a second (52) and third (53) substantially annular element aligned with each other along said axis (9), and having an outside diameter smaller than the inside diameter of said beads (4); two actuating devices (28, 29) being connected to a respective one of said two rings (25, 26) of supporting members (27), for moving the respective ring (25)(26) of supporting members (27) between said operating position and an idle position, wherein said two rings (25, 26) of supporting members (27) have an outside diameter smaller than the inside diameter of said beads (4).

2. A unit as claimed in Claim 1, characterized by the fact that each said number (48)(49) of segments (23) comprises alternate segments (23) of said first annular element (24).

3. A unit as claimed in Claim 1 or 2, characterized by the fact that, for each respective segment (23), each said transmission (50)(51) comprises an articulated parallelogram (54) in turn comprising at least one connecting rod (55) defined by the segment itself (23), and at least one crank (57); and an activating device (59) connected to said crank (57), for moving said connecting rod (55) between said idle and operating positions.

4. A unit as claimed in Claim 3, characterized by the fact that said activating device (59) comprises a cam (65), and a tappet member (60, 64) connected to said crank (57).

5. A unit as claimed in Claim 4, characterized by the fact that said cam (65) is axially fixed along said axis (9).

5 6. A unit as claimed in Claim 5, characterized by the fact that each said articulated parallelogram (54) comprises a frame (56) movable parallel to said axis (9); drive means (11) being connected to said activating device (59), for moving said frame (56) along said axis (9) and between a withdrawn idle position and a forward operating position; and said tappet member (60, 64) being movable with said frame (56) along said axis (9).

10 7. A unit as claimed in Claim 6, characterized by the fact that the tappet member (60, 64) comprises a rocker arm (60) pivoting on said frame (56) and in turn comprising a first and second arm; the first arm being connected to said cam (65); and transmission means (62, 63) being provided for connecting the second arm to said crank (57).

15 8. A unit as claimed in Claim 7, characterized by the fact that said transmission means (62, 63) comprise first teeth (62) formed on the second arm, about the pivot (61) of said rocker arm (60); and second teeth (63) formed about the pivot of said crank (57) on said frame (56); the second teeth (63) being integral with the crank (57), and being rotated by the first teeth (62) about said pivot of said crank (57) when the frame (56) is moved along said axis (9) by said drive means (11).

20 9. A unit as claimed in Claim 6, 7 or 8, characterized by the fact that said drive means (11) comprise a motor (8) for rotating said first annular supporting element (24) and said rings (25, 26) of supporting members (27) about said axis (9).

25 10. A unit as claimed in any one of the foregoing Claims, characterized by the fact that actuating means (41, 45) are provided for moving said rings (25, 26) of supporting members (27) along said axis (9).

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#### Patentansprüche

1. Innere Halteeinheit für die Herstellung einer toroidalen Karkasse (2), umfassend einen ringförmigen Mittelteil (6), zwei Wülste (4) auf beiden Seiten des ringförmigen Mittelteils (6) und zwei Seitenwände (5), von denen jede durch eine unter Verwendung von wenigstens einem Cord gebildete Aufeinanderfolge von Schlingen bzw. Maschen definiert ist und eine jeweilige Wulst (4) umschließt; wobei die Einheit (1) eine Anzahl von Ringsegmenten (23) umfaßt, die in eine Betriebsposition bewegbar sind, worin jedes Segment (23) zwei benachbarte Seg-

mente (23) kontaktiert und damit abgefluchtet ist, um ein ersts ringförmiges Halteelement (24) für den ringförmigen Mittelfeil (6) der toroidalen Karkasse (2) zu definieren; und zwei Ringe (25, 26) aus Halteteilen (27) auf beiden Seiten des ersten ringförmigen Elements (24), die koaxial zueinander und zu der Achse (9) des ersten ringförmigen Elements (24) sind und welche in eine Betriebspotition zum Halten der beiden Wüste (4) bewegbar sind; wobei das erste ringförmige Halteelement (24) radial auswärts von und axial zentriert in bezug zu den beiden Ringen (25, 26) aus Halteteilen (27) in der Betriebspotition lokalisiert ist, so daß es mit denselben zwei ringförmige Öffnungen begrenzt; dadurch gekennzeichnet, daß das erste ringförmige Element (24) wenigstens eine erste (48) und zweite (49) Anzahl der Segmente (23) umfaßt; wobei eine erste (50) und zweite (51) Kraftübertragung mit der ersten (48) bzw. zweiten (49) Anzahl von Segmenten (23) verbunden ist zum Bewegen der jeweiligen Anzahl (48) (49) von Segmenten (23) zwischen der Betriebspotition und einer Ruheposition, worin die beiden Anzahlen (48, 49) von Segmenten (23) jeweils ein zweites (52) und drittes (53) im wesentlichen ringförmiges Element definieren, die zueinander längs der Achse (9) ausgerichtet bzw. abgefluchtet sind und einen Außendurchmesser haben, der kleiner als der Innendurchmesser der Wüste (4) ist; wobei zwei Betätigungsseinrichtungen (28, 29) je mit einem der beiden Ringe (25, 26) von Halteteilen (27) verbunden sind zum Bewegen des jeweiligen Rings (25) (26) von Halteteilen (27) zwischen der Betriebspotition und einer Ruheposition, worin die beiden Ringe (25, 26) von Halteteilen (27) einen Außendurchmesser haben, der kleiner als der Innendurchmesser der Wüste (4) ist.

2. Einheit nach Anspruch 1, dadurch gekennzeichnet, daß jede Anzahl (48) (49) von Segmenten (23) alternierende bzw. miteinander abwechselnde Segmente (23) des ersten ringförmigen Elements (24) umfaßt.

3. Einheit nach Anspruch 1 oder 2, dadurch gekennzeichnet, daß für jedes jeweilige Segment (23) jede Kraftübertragung (50, 51) ein gelenkiges Parallelogramm (54) umfaßt, das seinerseits wenigstens eine Verbindungsstrebe bzw. einen Verbindungsbügel (55), die bzw. der durch das Segment (23) selbst definiert ist, und wenigstens eine Kurbel bzw. einen Leitarm (57) umfaßt; und eine mit der Kurbel bzw. dem Leitarm (57) verbundene Aktivierungseinrichtung (59) zum Bewegen der Verbindungsstrebe bzw. des Verbindungsbügels (55) zwischen der Ruhe- und Betriebspotition.

4. Einheit nach Anspruch 3, dadurch gekennzeichnet, daß die Aktivierungseinrichtung (59) eine

Nockenbahn (65) und ein mit der Kurbel bzw. dem Leitarm (57) verbundenes Nockenteil (60, 64) umfaßt.

5. Einheit nach Anspruch 4, dadurch gekennzeichnet, daß die Nockenbahn (65) axial längs der Achse (9) fixiert ist.

6. Einheit nach Anspruch 5, dadurch gekennzeichnet, daß jedes gelenkige Parallelogramm (54) einen Rahmen (56) umfaßt, der parallel zu der Achse (9) bewegbar ist; wobei Antriebsmittel (11) mit der Aktivierungseinrichtung (59) verbunden sind zum Bewegen des Rahmens (56) längs der Achse (9) und zwischen einer zurückgezogenen Ruheposition und einer vorwärtigen Betriebspotition; und das Nockenteil (60, 64) mit dem Rahmen (56) längs der Achse (9) bewegbar ist.

10. Einheit nach Anspruch 6, dadurch gekennzeichnet, daß das Nockenteil (60, 64) eine sich auf dem Rahmen (56) drehende bzw. verschwenkende Schwinge (60) umfaßt, die ihrerseits einen ersten und zweiten Arm umfaßt; wobei der erste Arm mit der Nockenbahn (65) verbunden ist; und wobei Kraftübertragungsmittel (62, 63) zum Verbinden des zweiten Arms mit der Kurbel bzw. dem Leitarm (57) vorgesehen sind.

15. Einheit nach Anspruch 7, dadurch gekennzeichnet, daß die Kraftübertragungsmittel (62, 63) erste Zähne (62), die auf dem zweiten Arm um die Drehachse (61) der Schwinge (60) ausgebildet sind; und Zweite Zähne (63), die um die Drehachse der Kurbel bzw. des Leitarms (57) auf dem Rahmen (56) ausgebildet sind, umfassen; wobei die zweiten Zähne (63) integral mit der Kurbel bzw. dem Leitarm (57) sind und durch die ersten Zähne (62) um die Drehachse der Kurbel bzw. des Leitarms (57) gedreht werden, wenn der Rahmen (56) längs der Achse (9) durch die Antriebsmittel (11) bewegt wird.

20. Einheit nach Anspruch 8, dadurch gekennzeichnet, daß die Kraftübertragungsmittel (62, 63) erste Zähne (62), die auf dem zweiten Arm um die Drehachse (61) der Schwinge (60) ausgebildet sind; und Zweite Zähne (63), die um die Drehachse der Kurbel bzw. des Leitarms (57) auf dem Rahmen (56) ausgebildet sind, umfassen; wobei die zweiten Zähne (63) integral mit der Kurbel bzw. dem Leitarm (57) sind und durch die ersten Zähne (62) um die Drehachse der Kurbel bzw. des Leitarms (57) gedreht werden, wenn der Rahmen (56) längs der Achse (9) durch die Antriebsmittel (11) bewegt wird.

25. Einheit nach Anspruch 6, 7 oder 8, dadurch gekennzeichnet, daß die Antriebsmittel (11) einen Motor (8) zum Drehen des ersten ringförmigen Halteelements (26) und der Ringe (25, 26) der Halteteile (27) um die Achse (9) umfassen.

30. Einheit nach irgendeinem der vorhergehenden Ansprüche, dadurch gekennzeichnet, daß Betätigungsmitte (41, 45) vorgesehen sind zum Bewegen der Ringe (25, 26) aus Halteteilen (27) längs der Achse (9).

35. Revendications

1. Dispositif de support intérieur pour la production

d'une carcasse toroïdale (2) comprenant une partie annulaire intermédiaire (6), deux talons (4) situés sur l'un et l'autre côtés de la partie annulaire intermédiaire (6) et deux parois latérales (5), chacune étant constituée d'une succession de boucles formées à l'aide d'au moins une corde et enfermant un talon respectif (4) ; le dispositif (1) comprenant plusieurs segments annulaires (23) déplaçables pour être mis à une position de fonctionnement à laquelle chaque segment (23) est en contact avec, et aligné sur, deux segments voisins (23) pour constituer un premier élément annulaire (24) de support de la partie annulaire intermédiaire (6) de la carcasse toroïdale (2) ; et deux anneaux (25, 26) d'éléments de support (27) qui sont situés sur l'un et l'autre côtés dudit premier élément annulaire (24) et sont coaxiaux l'un à l'autre et à l'axe (9) du premier élément annulaire (24), lesdits deux anneaux étant déplaçables pour être mis à une position de fonctionnement pour supporter lesdits deux talons (4) ; ledit premier élément annulaire de support (24) étant placé radialement à l'extérieur des, et centré axialement par rapport auxdits, deux anneaux (25, 26) d'éléments de support (27) à ladite position de fonctionnement de manière à délimiter avec ceux-ci deux ouvertures annulaires ; caractérisé par le fait que le premier élément annulaire (24) comprend au moins un premier (48) et un deuxième (49) nombres desdits segments (23) ; une première (50) et une deuxième (51) transmissions étant reliées respectivement audit premier (48) et audit deuxième (49) nombres de segments (23) pour déplacer les nombres respectifs (48) (49) de segments (23) entre ladite position de fonctionnement et une position de repos à laquelle lesdits deux nombres (48, 49) de segments (23) constituent respectivement un deuxième (52) et un troisième (53) éléments sensibles annulaires alignés l'un sur l'autre le long dudit axe (9) et ayant un diamètre extérieur plus petit que le diamètre intérieur desdits talons (4) ; deux dispositifs d'actionnement (28, 29) étant reliés à l'un respectif desdits deux anneaux (25, 26) d'éléments de support (27) pour déplacer les anneaux respectifs (25) (26) d'éléments de support (27) entre ladite position de fonctionnement et une position de repos à laquelle lesdits deux anneaux (25, 26) d'éléments de support (27) ont un diamètre extérieur plus petit que le diamètre intérieur desdits talons (4).

2. Dispositif selon la revendication 1, caractérisé par le fait que chacun desdits nombres (48) (49) de segments (23) comprend des segments alternés (23) dudit premier élément annulaire (24).
3. Dispositif selon la revendication 1 ou 2, caractérisé par le fait que, pour chaque segment respectif (23), chacune desdites transmissions (50) (51) com-

prend un parallélogramme articulé (54) comprenant de son côté au moins une tige de liaison (55) constituée dudit segment lui-même (23), ainsi qu'au moins une manivelle (57) ; et un dispositif de mise en fonction (59) relié à ladite manivelle (57) pour déplacer ladite tige de liaison (55) entre lesdites positions de repos et de fonctionnement.

4. Dispositif selon la revendication 3, caractérisé par le fait que ledit dispositif de mise en fonction (59) comprend une came (65) et un élément de poussée (60, 64) relié à ladite manivelle (57).
5. Dispositif selon la revendication 4, caractérisé par le fait que ladite came (65) est axialement fixe le long dudit axe (9).
6. Dispositif selon la revendication 5, caractérisé par le fait que chacun desdits parallélogrammes articulés (54) comprend un châssis (56) mobile parallèlement audit axe (9) ; des moyens de commande (11) étant reliés audit dispositif de mise en fonction (59) pour déplacer ledit châssis (56) le long dudit axe (9) et entre une position de repos en retrait et une position de fonctionnement avancée ; et ledit élément de poussée (60, 64) étant mobile avec ledit châssis (56) le long dudit axe (9).
7. Dispositif selon la revendication 6, caractérisé par le fait que l'élément de poussée (60, 64) comprend un bras oscillant (60) pivotant sur ledit châssis (56) et comprenant de son côté un premier et un deuxième bras ; le premier bras étant relié à ladite came (65) ; et des moyens de transmission (62, 63) étant prévus pour relier le deuxième bras à ladite manivelle (57).
8. Dispositif selon la revendication 7, caractérisé par le fait que lesdits moyens de transmission (62, 63) comprennent des premières dents (62) réalisées sur le deuxième bras, autour du pivot (61) dudit bras oscillant (60) ; et des deuxièmes dents (63) formées autour dudit pivot de montage de ladite manivelle (57) sur ledit châssis (56) ; les deuxièmes dents (63) étant monobloc avec la manivelle (57) et étant entraînées en rotation par les premières dents (62) autour dudit pivot de ladite manivelle (57) lorsque le châssis (56) est déplacé le long dudit axe (9) par lesdits moyens de commande (11).
9. Dispositif selon la revendication 6, 7 ou 8 caractérisé par le fait que lesdits moyens de commande (11) comprennent un moteur (8) destiné à faire tourner ledit premier élément annulaire de support (24) et lesdits anneaux (25, 26) desdits éléments de support (27) autour dudit axe (9).

10. Dispositif selon l'une quelconque des revendications précédentes, caractérisé par le fait que des moyens d'actionnement (41, 45) sont prévus pour déplacer lesdits anneaux (25, 26) d'éléments de support (27) le long dudit axe (9).  
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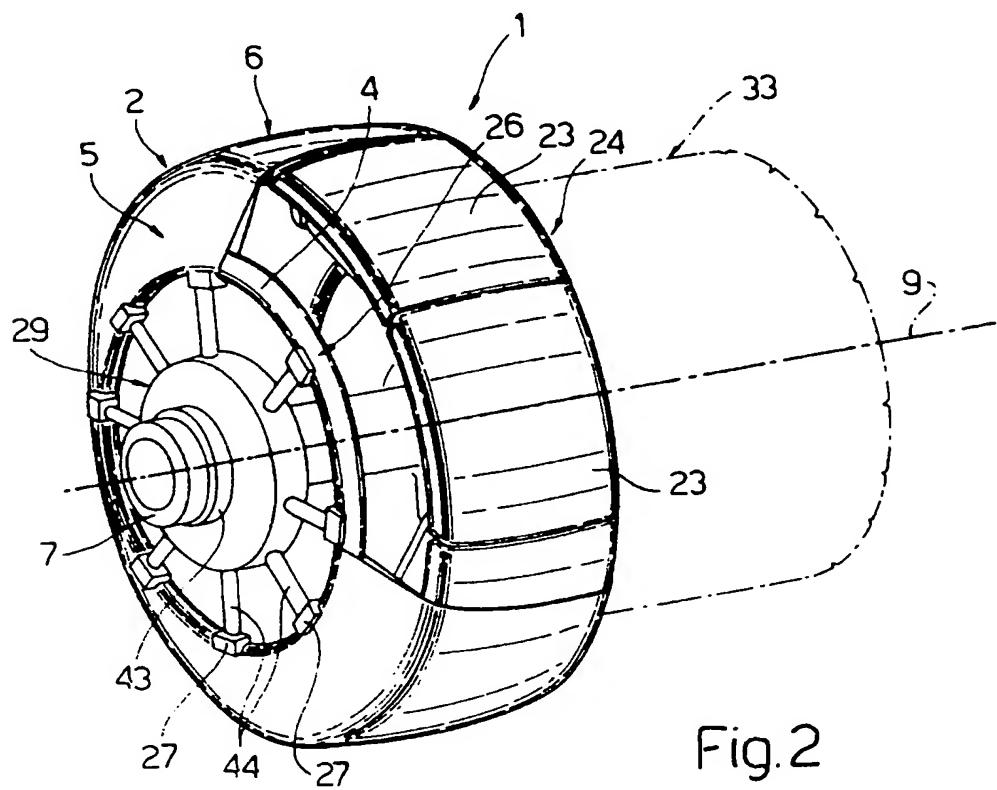
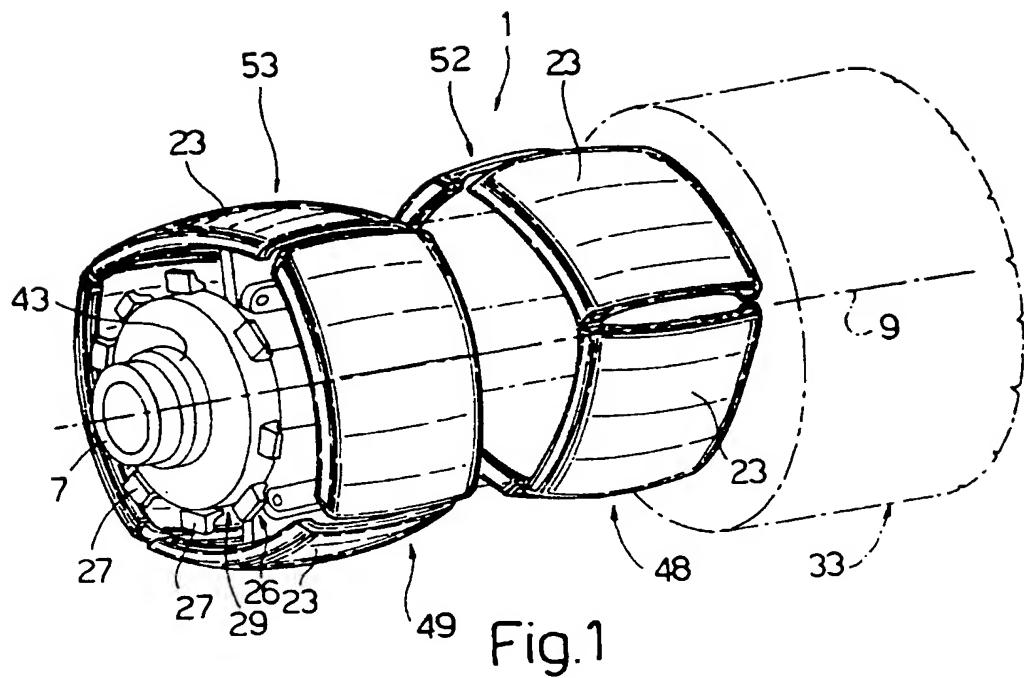
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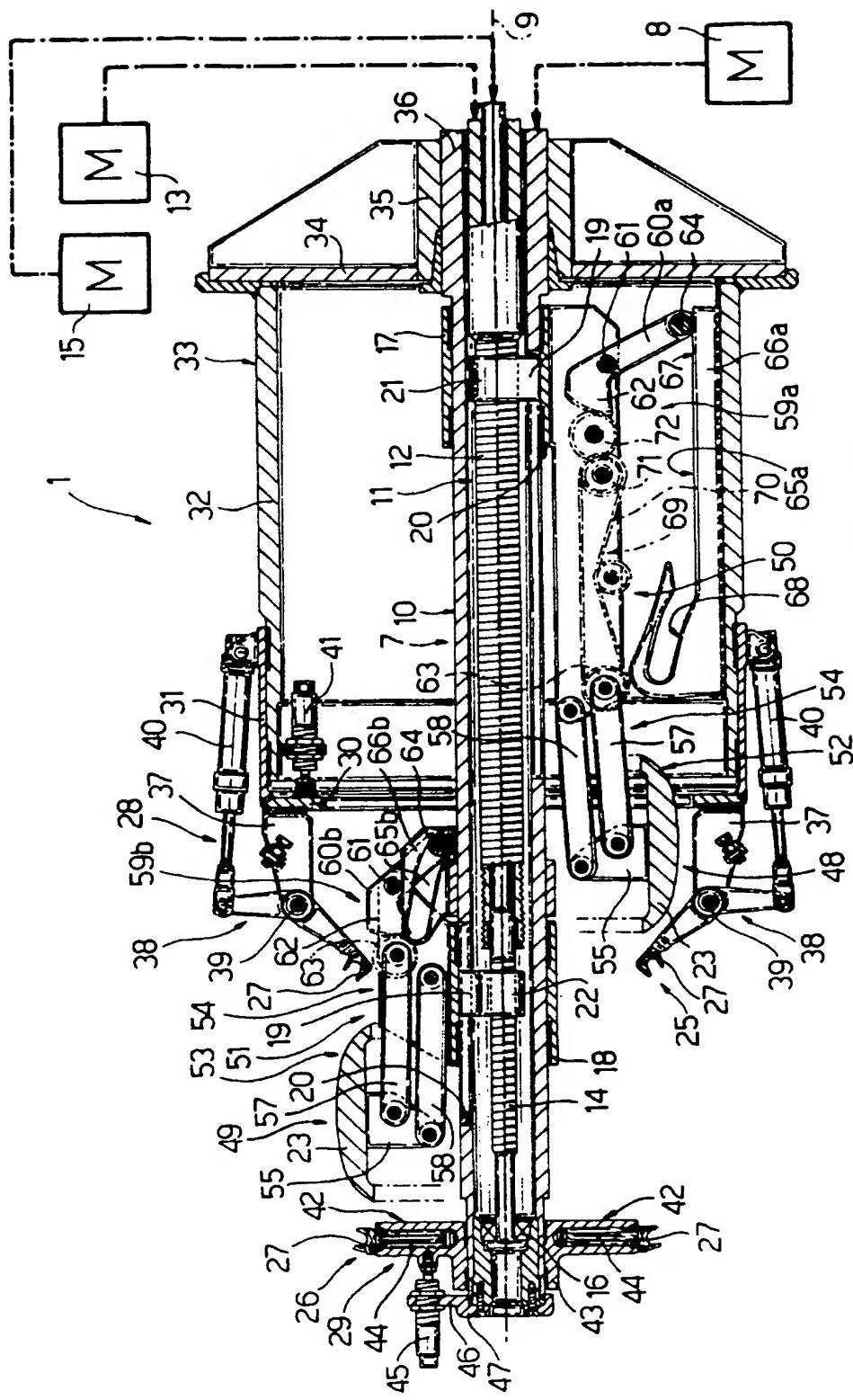
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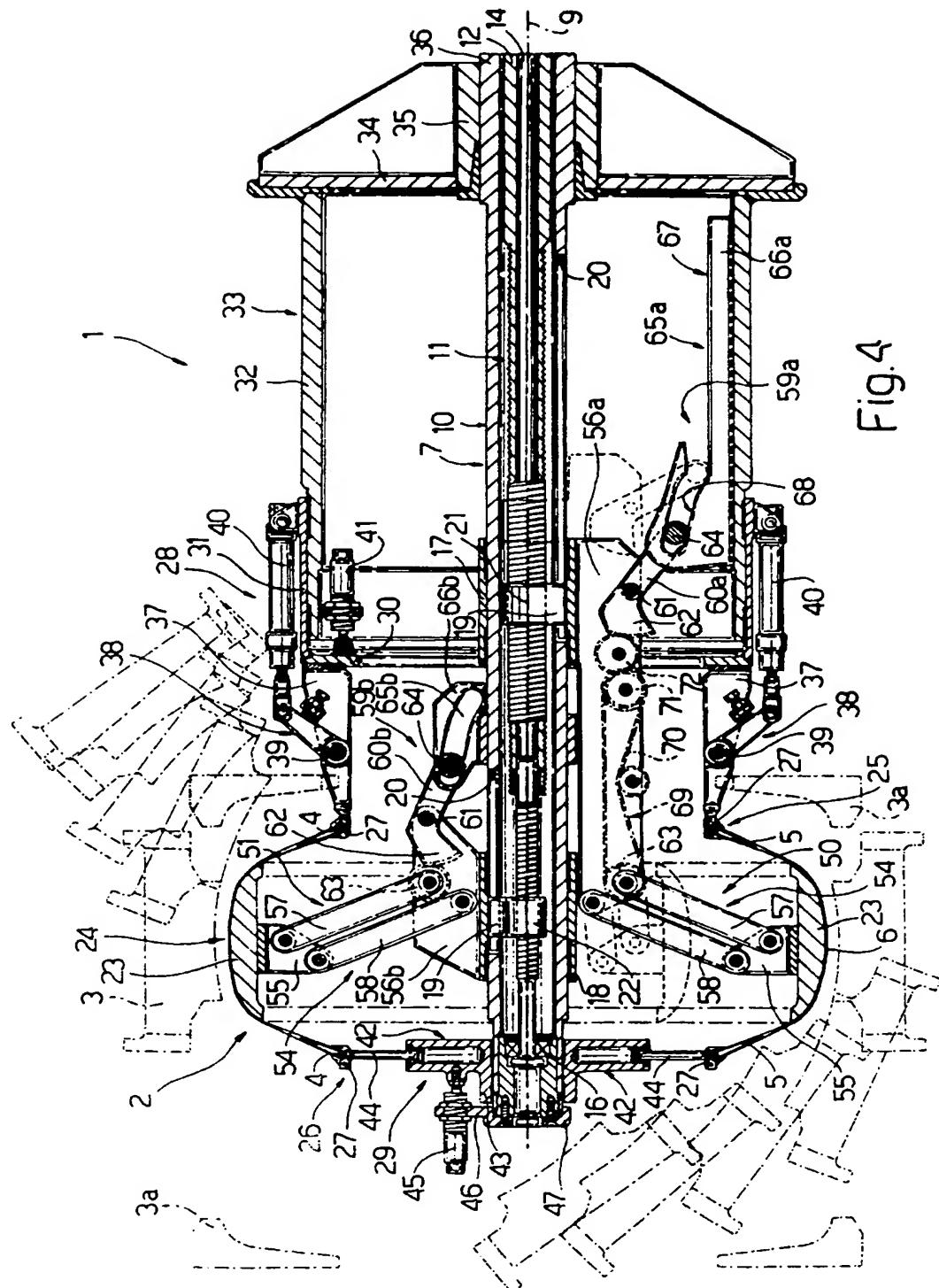


Fig. 4

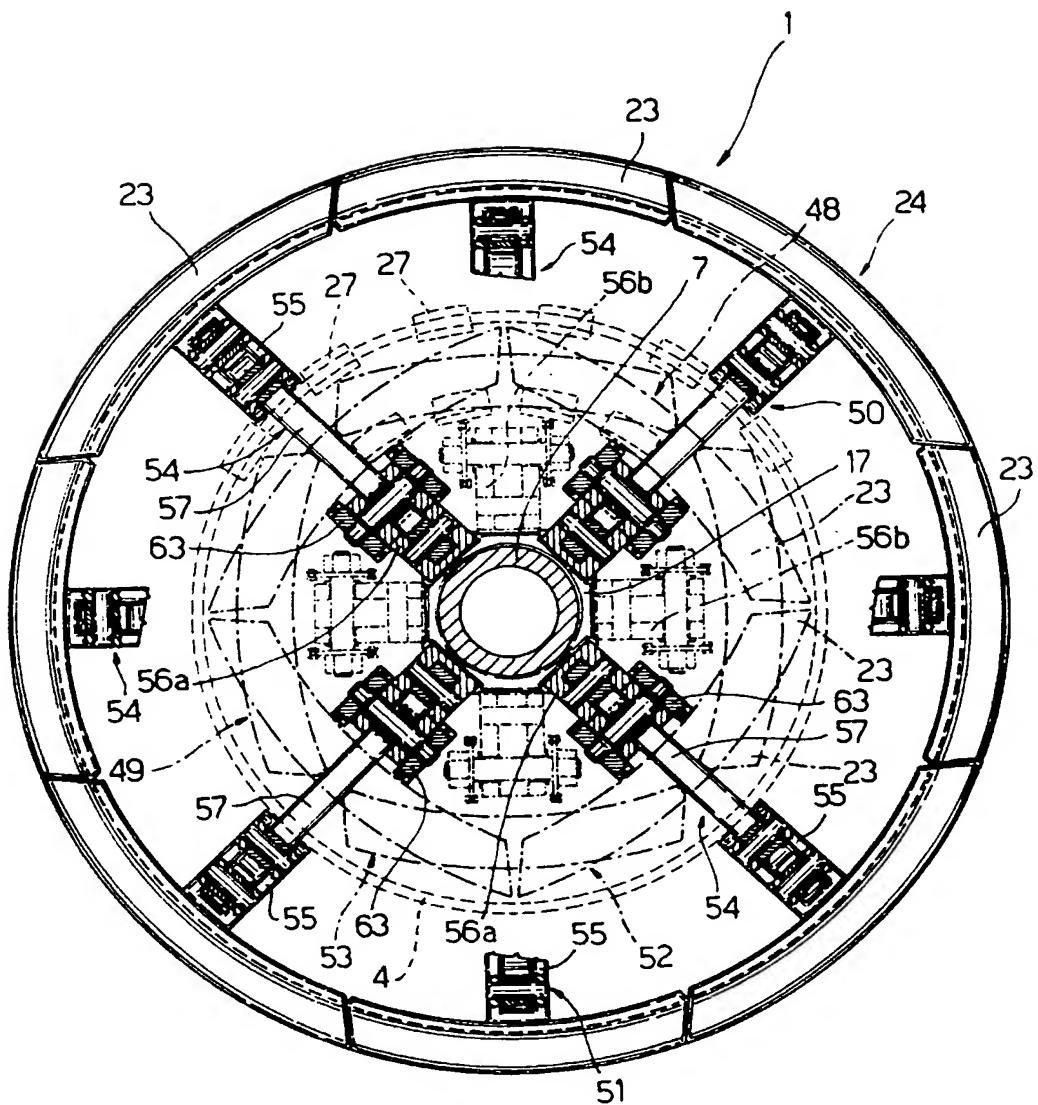


Fig.5